

## KISSsoft Release 2024 Corrections, Features and Improvements

### Service Pack 4

#### General

##### SP 4 - **Corner distance calculation for not sharp-edged abutment in snap-ring**

The corner distance  $h$  was calculated without influence of the corner distance  $g$ , when the abutment is not sharp-edged.

##### SP 4 - **Helical gears right handed weren't fully generated anymore.**

Helical gears right handed weren't fully generated anymore.

##### SP 4 - **Own input material data for snap ring**

Own input material data for snap ring is now saved correctly.

#### System Module General

##### SP 4 - **3D section view**

Improvement in the 3D-viewer section view in the Z-Y plane.

##### SP 4 - **3D viewer**

Improvement in the simplified gear geometry used in the 3D view of gears with multiple contacts.

##### SP 4 - **Animation of the rotation of bearings in the the 3D viewer**

The outer ring of connecting bearings follows the rotation of the outer shaft.

##### SP 4 - **Bearing and shaft auxiliary results**

The bearing and shaft auxiliary results now consider planetary and differential copies in the auxiliary results.

##### SP 4 - **Correction in the selection of shafts to show in the system modal analysis plot**

Only the shafts of the selected shaft calculation are now shown in the system modal analysis plot.

##### SP 4 - **Gear facewidth used in the 3D viewer**

Correction in the gear face-width used in the 3D viewer, for the case of multi-mesh gears.

##### SP 4 - **Handling of messages from sub-calculations**

Correction in the handling of messages of sub-calculations in a system module calculation.

##### SP 4 - **Include face gears in system backlash calculation**

System backlash calculations now includes also face gears.

##### SP 4 - **Interference check**

Improved interference check between gears and housing elements.

## Housing Deformation

### SP 4 - Housing deformation calculation

Improvement in the update of shaft calculations in the iterations of housing deformation.

### SP 4 - Housing deformation special tab

Update of the housing deformation tab, after loading a file.

### SP 4 - Nastran symmetric reduced stiffness matrix

Improvement in the reading of symmetric Nastran reduced stiffness matrices.

### SP 4 - Reading of initial offsets from reduced stiffness matrix file

Improvement in the stability of the initial offsets reader.

## Eigenfrequencies

### SP 4 - 3D view of mode shapes

Improvement in the animation of mode shapes in the 3D viewer.

### SP 4 - Better format in reports

The format of the reports in modal analysis and Campbell diagram is improved.

### SP 4 - Campbell diagram analysis

Improvement in the Campbell diagram analysis in shaft calculations for some special cases, where the number of matching eigen-frequencies is not the same for all speeds.

### SP 4 - Simultaneous change of UI parameters, and graphics property list in the Campbell diagram

Simultaneous change of UI parameters, and graphics property list in the Campbell diagram is not possible. For instant, one can see the issue by changing "Number of calculations steps" in the UI, and "Number of resonance curve" in the property list.

## Forced Response Shafts

### SP 4 - Curve saving in the forced response analysis in the shaft module

Activation of saving the curves in the forced response analysis graphic in the shaft module, when the option "Reference position" is selected.

## Gears

### SP 4 - Language updated in the modification table

The language of the modification type in the modification table is updated instantaneously.

### SP 4 - Pressure viscosity coefficient for PAG-based oils

The calculation has been improved to use the formula from ISO/TR 6336-22:2018.

### SP 4 - Sizing of the tip gear alteration

Sizing of the tip alteration in tab "Reference profile" now sets the calculation to inconsistent.

## Cylindrical Gears Rating

### SP 4 - Annex E calculation with modifications per flank

Annex E calculation with modifications per flank now selects correct flank when single bin from load spectrum is selected.

### SP 4 - Annex E planet shaft deformation

Corrected shaft deformations during Annex E calculations for separate planets.

## Cylindrical Gears Geometry

### SP 4 - 3D export of cylindrical, planetary gears and racks

Correction in the orientation of the gears in the 3D export.

### SP 4 - Backlash calculation with actual tooth form

Flank line modifications for the first gear in a gear pair are now properly considered in the backlash calculation with actual tooth form.

## Bevel Gears

### SP 4 - Bevel Annex E with multimesh

Bevel face load factor according to Annex E can be calculated when the bevel has multiple meshes.

### SP 4 - Warning message for factor $Z_{KP}$

The warning message for factor  $Z_{KP}$  is now shown correctly in case of user defined value.

## Globoid Worm Gears

### SP 4 - Manufacturing data for worm wheel

Measurement over 3 pins for the worm wheel in the manufacturing data report now has the correct value.

## Beveloid Gears

### SP 4 - System 3D view of beveloid gears

Fix in the overlap of beveloid gears when switching to "Position like an internal toothing" and back.

## Shafts

### SP 4 - Gear inner diameter

When gear data is read from the gear calculation file, inner diameter of the gear (outer diameter in case of internal gear) is now also read from the file.

### SP 4 - Rotational constraint of the shaft for single bin calculations

In case of single bin calculations rotational constraint of the shaft is now properly set.

### SP 4 - Shaft strength calculation with load spectra and small load sum

For the shaft strength calculation with load spectra and small load sum,  $K_{BK}$  factor will be set to 1.

## Bearings

### SP 4 - Pressure curve graphic for roller bearings

Roller bearing pressure curve graphic now shows different colors and line styles for each roller.

## Bolts

### SP 4 - For the length of engagement $m_{ges}$

Value for the length of engagement  $m_{ges}$  is now kept, if the checkbox is activated.

### SP 4 - Roughness of the plates read from database

The roughness of the plates is read out from the database correctly, and the variables are updated.

### SP 4 - Tightening factor influence on $F_{V1}$

The tightening factor now correctly affects the calculated  $F_{V1}$ .

### SP 4 - Tolerance 7H and 8H for $D_{1max}$

The tolerances for 7H and 8H for  $D_{1max}$  have the proper order in the database.

## FEM Calculations

### SP 4 - Plotting of Tresca equivalent stress

The Tresca equivalent stress is now plotted to the graphic results of FEM components.

## Scripting

### SP 4 - Array literals

Variables can now be initialized from array literals, e.g., `number[][] array = [ [0, 1, 2], [2, 3, 4] ]`

### SP 4 - Order of indices

When several indices are present in a variable, the correct order is applied.

### SP 4 - Unknown variable message

An error message is shown in the script output if an unknown variable is found.

## STEP-Interface

### SP 4 - Step export of shaft system with worm gear

Improvement in the orientation of a worm gear, contained in the step export of a shaft system.

## CAD-Interfaces

### SP 4 - Interface to Autodesk Inventor 2026

Interface to Autodesk Inventor 2026 added.

## KISSsys

### SP 4 - 3DGLViewer shaft elements user color selection

The 3DGLViewer shaft elements show the user color selection.

## Service Pack 3

### General

#### SP 3 - File format for saved tables

File format for saving tables set to UTF16 LE BOM.

#### SP 3 - Initialization with start parameter

Parameter initialization with START parameter aligned with standard initialization.

#### SP 3 - Torsional stress in report now includes the reduction coefficient

The formula for torsional stress in VDI2230 includes the reduction coefficient. Therefore we also have to include this coefficient in the amount we show in the report. This means the amount now reflects the torsional stress under operating conditions, not during mounting.

## System Module General

### SP 3 - Churning losses calculation

Churning losses calculation has been improved to better consider bevel gears and immersion depth for different axes orientations.

### SP 3 - Connecting bearing tables

Bearing tables now show the correct information for connecting elements.

### SP 3 - Data representation table shaft overview

Data access for data representation table shaft overview fixed.

### SP 3 - Default values in gear load spectrum table

Unspecified factors and values in the load spectrum table will have a default value if not specified in order to avoid cases where a 0 was used and causing an error.

### SP 3 - Force transfer for planet copies

Force transfers for planet copies are now correctly considering the number of planets when transferring any support reaction force to the receiver load on the carrier.

### SP 3 - Over defined kinematics

Over defined kinematics are now converging during any step of the modelling phase.

### SP 3 - Roll, polar and meshing angles

Roll, polar, and meshing angles are now defined when a gear is positioned according to two other fixed gears.

### SP 3 - Shaft operating temperature in load spectrum

The shaft operating temperature can now be defined for each individual shaft in the system load spectrum.

### SP 3 - State of submodule graphics

The state of the graphic windows of all submodules of a system calculation is saved and restored automatically.

### SP 3 - Updated profile shift coefficient in gear list report

Profile shift coefficient in the gear list report no longer shows 0 in some cases with a defined profile shift.

## Sketcher

### SP 3 - Sketcher label positioning

The labels can now be positioned closer to the shafts and other shaft elements.

## Eigenfrequencies

### SP 3 - Eigenfrequencies in a system of shafts

Improvement in the search for eigenfrequencies in a system of shafts.

### SP 3 - System characteristic frequencies diagrams enabled with the correct license

The diagrams for the system's characteristic frequencies are activated under the correct licensing.

## Forced Response Shafts

### SP 3 - Time domain data in the forced response analysis

Calculation of the excitation frequencies and time domain data in the forced response analysis of the shaft calculation module (when the reference speed is selected), has been updated to consider the excitation frequencies of the unbalanced masses.

## Cylindrical Gears Rating

### SP 3 - Addition of error message in cases where the gear body deformation fails during the Annex E calculation

If the gear body stiffness has not been able to be included to the Annex E calculation an error message appears to the user.

### SP 3 - Annex E results with matching load spectrums

When Annex E is run with shaft files that having a matching load spectrum to the gear calculation, each bin is correctly used for the load spectrum results. This also occurs for the contact analysis results.

### SP 3 - Face load factor report tables

Total deformation in the gap tables now includes the gear body deformation and references to "gb" are replaced with "FE".

### SP 3 - Micropitting single or nominal load calculation

Improvements have been made to ensure the hidden load spectrum application factor setting is no longer considered when calculating single or nominal load. Previously, this issue caused the local Hertzian pressure to be calculated as zero.

### SP 3 - Sizing of torque or power with Annex E

Accuracy of the torque or power sizing functionality using Annex E is improved.



## Cylindrical Gears Contact Analysis

### SP 3 - Local stiffness curve for planetary contact analysis considers friction properly

The tangent stiffness curve (operating stiffness point) now considers the friction correctly.

### SP 3 - Normal force graphic improvement

The normal force along pitch diameter graphic no longer gives an error when Gear 2 is driving.

### SP 3 - Sound pressure level based on Masuda

The sound pressure level based on Masuda now gives the same results if you choose transmission error setting as an angle instead of a distance.

## Cylindrical Gears Geometry

### SP 3 - Calculation of $d_{Fa}$ with rounding in axial section

Calculation of  $d_{Fa}$  with applied rounding in axial section of 0.0 mm now returns correct values.

### SP 3 - Marker value conversion according to the coordinate unit settings

Marker values are now shown correctly according to the coordinate unit settings.

### SP 3 - Modifications for asymmetric gears

In cases where modifications were applied for asymmetric gears and the tooth form had an error, the flanks of the modifications might get swapped.

### SP 3 - Profile diagram for asymmetric gears

Profile diagram of asymmetric gears is now shown correctly for the left flank.

### SP 3 - Sizing profile shift

Using the sizing button for the profile shift coefficient was sometimes overwriting the user defined value for "Start modification at height coefficient".

### SP 3 - Start of modification for asymmetric gears

Parameter "Modification value defined at" is now saved correctly to the calculation file for asymmetric gears.

### SP 3 - Twist applied through tab Tooth form

Twist applied through the tooth form operation "Take modifications into account" is now considered correctly in the 3D models and contact analysis.

## Bevel Gears

### SP 3 - Contact analysis results window

Contact analysis results for bevel gears are shown correctly in the result window when the calculation method is chosen as static or differential.

### SP 3 - Own input of face load factor $K_{H\beta}$

Face load factor  $K_{H\beta}$  when set to "Own input" is now correctly considered in the AGMA 2003 calculation.

## Shafts

### SP 3 - Bevel gear in shaft editor

Bevel gears that are not fully defined are displayed in the shaft editor anyway.

### SP 3 - Outer contour table data input

Outer contour table allows input or roughness data.

### SP 3 - Update issue in shaft editor

Cross sections in shaft editor do repaint after each calculation.

## Bearings

### SP 3 - Rolling bearing clearance change calculation

Housing temperature was not considered in the rolling bearing clearance change calculation.

### SP 3 - Rolling bearing damage with load spectrum and lifetime acc. to ISO 281

Precision of rolling bearing damage calculation in case of lifetime acc. to ISO 281 and for load spectrum with bin frequency < 0.001% has been improved.

## Shaft-Hub Connections

### SP 3 - Change of minimum effective tolerance value $S_{vmin}$ in DIN 5480

The minimum effective tolerance value  $S_{vmin}$  was not defined in DIN 5480. Now the definition from the ISO 4156 is used to calculate the value.

## Bolts

### SP 3 - Typo in guideline VDI 2230 paper 1, equation 213

In the guideline VDI 2230 paper 1, equation 213 was written in the denominator  $d_{min}$ , however it should be  $D_{1max}$  (according Prof. Wuttke, VDI committee). The correct equation is now implemented.

## FEM Calculations

### SP 3 - 2D FEM graphic display in case of helical gears

In case of helical gear geometries the 2D FEM analysis solves a 2D model of the equivalent spur gear. In the 2D graphic the diameter of the equivalent tooth form is displayed (as in version <=23).

### SP 3 - Export filter of FEM graphics

The export options of FEM graphics have been adjusted based on the graphic type (geometry, mesh, results).

### SP 3 - FEM root stress 2D options for old files

When loading old KISSsoft files containing 2D/3D FEM root stress calculations the displayed FEM options were not always up to date. This has been corrected now.

### SP 3 - Finit-Elements mesh of gear bodies with thick rim

In case of a gear body with thick rim defined the FE algorithm has been improved to include more elements radially.

### SP 3 - Robustness improvement of gear body FEM model creation

The creation of the FEM model of the gear body has been considerably improved in case of manual definition of its cross section.

### SP 3 - Stress singularity display of 3D root stress calculations

The filtering algorithm for the stress singularity positions of the root stress 3D calculations has been improved.

## COM-Interface

### SP 3 - COM registration problem

The COM interface dependent on version can't register correctly for standalone versions (copy of the installation).

## CAD-Interfaces

### SP 3 - Interface to Siemens NX 2412

Interface to Siemens NX 2412 added.

### SP 3 - Interface to SolidEdge 2025

Interface to SolidEdge 2025 added.

### SP 3 - Interface to SolidWorks 2025

Interface to SolidWorks 2025 added.

## KISSsys

### SP 3 - Shafts files for bevel gear calculation

Shafts files for bevel gear calculations are now directly created when running the full model strength.

## Service Pack 2

### General

#### SP 2 - 3D section view

Improvement in the view of the 3D sections in the 3D viewer.

#### SP 2 - DUI example files extended

The DUI examples files have been extended to use the improved title handling.

#### SP 2 - Importing projects from previous releases

KISSsoft now supports importing projects from the previous releases 2023 back until 03-2016.

#### SP 2 - Message added before deleting table data

A message has been added to confirm if the user wants to delete the table data.

#### SP 2 - Messages with sign @

Messages or other information containing sign @ are now supported.

#### SP 2 - Overlapping texts in graphic properties removed

Overlapping texts in drop down list in graphic properties have been removed.

#### SP 2 - Tag fixed for KUI files

Tag fixed for all KUI files.

#### SP 2 - Units added for length per force

Units mm/N and mm/kN added.

#### SP 2 - User defined variables created with file load

User defined variables in a calculation file are created directly on load. Therefore they will be available without any user interface.

## System Module General

### SP 2 - Auxiliary results for boundaries attached to loads other than couplings

The generation of auxiliary results is supported when boundaries are attached to loads besides couplings.

### SP 2 - Bearing data in tables

Bearing data was swapped in the tables for some cases.

### SP 2 - Correction in the view of gear bodies in the 3D viewer

Correction in the view of gear bodies in the 3D viewer, when multiple gear bodies are visible.

### SP 2 - External lubrication ID check added

The software now checks if a file refers to an external lubrication ID and in such cases, switches to own input.

### SP 2 - Improve runtime of housing calculation

The runtime of the housing calculation is improved.

### SP 2 - Improvement in the position of the rings of bearings in the 3D viewer

The outer rings of bearings are now fixed in space and the inner rings follow the supporting shaft in the 3D viewer.

### SP 2 - Multiple fixed switchable elements on the same shaft

It is now possible to define multiple fixed switchable elements on the same shaft. The shaft calculation takes care of splitting the power accordingly.

### SP 2 - Planet copies in system

The possibility to create planet copies has been removed to simplify user experience.

### SP 2 - Planetary efficiency and loss results

Planetary efficiency and loss results are now displayed depending on the defined number of planets.

### SP 2 - Progress bar added in the interference check of the 3D viewer

A progress bar is added, showing the progress of calculation during an interference check in the 3D viewer.

### SP 2 - Rolling bearing damage for bearings without rotating speed

Bearing damage is now calculated for still-standing rolling bearings.

### SP 2 - Show critical element in gear pair safety factors in result overview

The result window now shows the gear pair safety factor for critical elements.

### SP 2 - Swap gear geometry

The geometry of two meshing gears can now be swapped on system level via the context menu. This action is only provided when possible.

### SP 2 - Switchable elements stiffness and clearance

Stiffness and clearance of Y-axis rotation can now be defined on system level for switchable elements.

## Sketcher

### SP 2 - Delete key for gear calculation activated

The delete key has been activated for multimesh gear calculations in the sketcher.

## Eigenfrequencies

### SP 2 - Modal analysis of the system module

Improvement in the positioning of the shafts' sketch in the modal analysis graphics of the system module,. Eigenmodes characterization is also improved.

## Forced Response Gears

### SP 2 - Collection of forced response data for the 3D view of the results

The collection of data to be seen in the forced response 3D results has been improved.

### SP 2 - Considering of gear mass in forced response based on the definition of "Gear on shaft" in shaft calculation has to be modified

Based on the definition of "Gear on shaft" in shaft calculation, consideration of the gear mass in the forced response is modified to properly model the effect of gear mass in calculating the system response.

### SP 2 - General plot in the forced response

Improvement in the handling of documentation point results in the general plot of the forced response.

## Gears

### SP 2 - Fine sizing center distance of three and four gear trains

Center distance in the fine sizing calculation of three and four gears is automatically taken from the tab "Basic data".

### SP 2 - Fine sizing graphic updates on delete

The fine sizing result graphic automatically updates after deleting results from the results table.

### SP 2 - Operating backlash calculation

The operating pressure angle is now considered more precisely in the operating backlash calculation.

### SP 2 - Root fatigue data for Delrin 51HSE

Root fatigue data is now available for Delrin 51HSE at 20°C, 80°C and 100°C.

## Cylindrical Gears Rating

### SP 2 - Annex E gear body deformation flank projection

When using gear body influence within the Annex E calculation, the projection for the gap is considered depending on the flank.

### SP 2 - Annex E shaft matcher for multimesh in three and four gear chain modules

The shaft matcher is improved during Annex E to better handle multimesh gears in shaft files.

### SP 2 - Ensure Annex E results use the correct load bin when shaft and gear have matching load spectrums

When Annex E is run with shaft files that have a matching load spectrum to the gear calculation, each bin is used for the load spectrum results. This also occurs for the contact analysis results.

### SP 2 - Import of the hardness curve

The user import of the hardness curve is now improved with an additional stability check.

### SP 2 - Optimization of $K_{H\beta}$ calculation logic in case of gear body consideration

In case of  $K_{H\beta}$  calculations including the effect of gear body, the FEM computation is now performed only at the beginning of the iterative calculation, thus reducing a lot the calculation time.

### SP 2 - Power calculation for double pinion planetary setup

Power, speed, and torque calculation are now calculated correctly for the double-pinion planetary setup.

## Cylindrical Gears Contact Analysis

### SP 2 - Contact lines on tooth flank color range

The color range in the "Contact lines on tooth flank" graphic now follows the values specified in the properties.

### SP 2 - Export of 3D contact analysis always shows all data

Sometimes, the export of .dat files from 3D contact analysis graphs cut off parts of the data. Export now includes all data shown in graphic.

### SP 2 - Improved stability of contact analysis with face chamfers

Contact analysis improved to be more stable in case of heavy face chamfers.

### SP 2 - Load contact pattern for internal gears

The no-load contact pattern shows the gap for internal gear meshes.

### SP 2 - Saving curves works again for contact analysis graphics

You can now save curves in 2D contact analysis graphics again.

### SP 2 - Small overhung edges effect improved on contact analysis results

When running the contact analysis with a small overhung edge, the edge influence has been improved.

## Cylindrical Gears Geometry

### SP 2 - Calculation of $d_{Ff}$ for premanufacturing

The root form diameter  $d_{Ff}$  for the premanufacturing is now calculated correctly for cases with undercut.

### SP 2 - Measurement over the balls for helical asymmetric gears

Calculation of measurement over the balls  $M_{dK}$  for helical asymmetric gears improved.

### SP 2 - Power skiving check with helix angle defined for the tool

Power skiving check when helix angle is defined for the tool is now calculated more precisely.

### SP 2 - Tooth form operation "Manufacture cylindrical gear with a gear generation process"

For tooth form operation "Manufacture cylindrical gear with a gear generation process" where the difference between  $s_{P0}^*$  and  $\pi/2$  is big ( $Abs(s_{P0}^* - \pi/2) > 0.25 \cdot \pi$ ), now the user defined  $s_{P0}^*$  factor is considered for the calculation of tooth form.

## Globoid Worm Gears

### SP 2 - Precision of the efficiency calculation

The efficiency calculation for DIN / ISO calculations of the globoid worm gears has been improved to be more precise. Also, the efficiency between the system (s020) module and the z170 module now matches.

### SP 2 - Sizing profile shift

The function to size profile shift (SizeProfileShift()) is now available through COM or skript.



## Shafts

### **SP 2 - 3D supports and plain bearings shaft position**

The 3D supports and plain bearings position is the center of the object.

### **SP 2 - 3D synchronizer color matches its icon color**

The 3D synchronizer color matches its icon turquoise color.

### **SP 2 - Improvement in the consideration of the shaft start position, in the 3D viewer of the W010 module**

The shaft position in the 3D viewer of the W010 module matches now correctly the "position in global system Y".

### **SP 2 - Improvement in the rotation animation in the 3D viewer**

Improvement in the view of the rotation animation, when a cross section is visible in the 3D viewer.

### **SP 2 - Improvement in the view of the coordinate system in the 3D viewer**

The coordinate system shown in the 3D viewer agrees now with the coordinate system shown in the shaft editor.

### **SP 2 - Messages in case if the sizing of the required lifetime can't be executed**

Messages are now shown if the sizing of the required lifetime can't be executed and the limit values for the lifetime is set.

### **SP 2 - Shaft editor selection with background drawing**

The selection of the inner contour has improved when a background drawing is loaded.

### **SP 2 - The equivalent stresses for static calculation**

The equivalent stresses for the neutral line and the bending line outside for the static calculation gave the same results. Now different components are used (case 1: shear stress = 0, case 2: bending stress = 0) to calculate the values.

### **SP 2 - Thermal expansion of housing in case of housing temperature load spectrum**

When calculating shafts with the load spectrum with defined housing temperature this temperature is now considered in the calculation.

### **SP 2 - Update of gear body in the shaft editor**

Improvement of the update of the gear body view in the shaft editor, based on a connected gear calculation.

## FEM Calculations

### SP 2 - 3D FEM results update after gear material modification

The 3D root stress result updates in case the user changes the material of a gear and performs the FEM calculation without repeating the LTCA calculation.

### SP 2 - Correction in the view of the gear body in the 3D graphic of the gear

The gear body was not shown in the 3D view of the gear after a gear body calculation.

### SP 2 - Correction of diameter display in the 3D tooth root stress calculation report

A typo displaying the radius instead of the diameter was corrected in the 3D root stress report.

### SP 2 - Equivalent gear diameter of helical gears in 2D FEM tooth root stress calculation

In case of helical gears, the 2D FEM is applied on an equivalent spur gear. After the calculation, all diameters should refer to the original helical gear, not the spur one.

### SP 2 - Improvement of 3D root stress sorting algorithm

The sorting algorithm for the calculation and reporting of the 3D tooth root stress was improved and made more robust.

### SP 2 - Improvement of the 3D FEM result default display functionality

In cases when a 3D FEM graphic was open and a re-calculation takes place the 3D graphic does not move back to the default state (showing the Principle Stress results). It stays on the result type the user had selected.

### SP 2 - Reading the gear body local path when a project is active

The local path definition when working with an active project is now supported. The STEP path of the gear body can be defined as a global or local path also when a project is active.

### SP 2 - Simplification of the export of stiffness matrix of gear body

In case the option to export the stiffness matrix of a gear body has been selected, the export will always overwrite any existing file.

### SP 2 - Update of FE meshing library to newest version

Improvement in the FE mesh functionalities mainly coming from an update of the external cm2-library to the new version 5.5.0.

## COM-Interface

### SP 2 - User interface initialization via COM

The user interface shows the default initialization when called via COM interface.

## STEP-Interface

### SP 2 - Improvement in the gears exported to a step file from the shaft calculation

Correction of the simplified gear geometry in the step export from the shaft calculation.

## CAD-Interfaces

### SP 2 - Interface Siemens NX 2406

Interface to Siemens NX 2406 added.

### SP 2 - Opening KISSsoft calculation file in CREO

It was not possible to reopen the KISSsoft calculation for Creo 9 and 10.

## KISSsys

### SP 2 - Color of lines in KISSsys 2D plots

Improvement of the color of lines in the 2D plots of KISSsys.

### SP 2 - Forced response calculation

The stability of the forced response calculation has been improved.

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Sharing Knowledge

## Service Pack 1

### General

#### SP 1 - Data representation tables extended

Data representation tables have been extended with headlines.

#### SP 1 - Helix angle cylindrical gears

Improvement in helix angle communicated to the shaft module for some cylindrical gears cases.

#### SP 1 - Permissible tensile stress $\sigma_{Mzul}$ calculation

The permissible tensile stress  $\sigma_{Mzul}$  is now calculated with the formula 143 according to VDI 2230 paper 1 (2015).

#### SP 1 - Shrinking diameters is now working in all cases

Shrinking diameters was not working in some cases, this is now fixed.

#### SP 1 - Spline calculations

Shaft-hub calculations are now performed with any given torque to allow the full system analysis.

#### SP 1 - Transparency of elements shown in a cut view of the 3D viewer

When a cut view is activated in the 3D viewer, the transparency of the elements shown is now updated correctly.

#### SP 1 - US customary units for data representation tables

Data representation tables show US customary units.

## System Module General

### SP 1 - Bevel gear tables extended

Values have been added to the data representation tables for bevel gears.

### SP 1 - Deleting all bins or columns

Deleting all bins or columns was not working for some old models, it is now fixed.

### SP 1 - Drawing number added to KISSdesign table

The shaft drawing number has been added to the data representation table in KISSdesign.

### SP 1 - Graphical representation of rotation arrows, torque and forces.

The scale and position of rotation arrows, torque and forces. is more accurate for models with very small gears.

### SP 1 - Hand of helix of a internal gear in 3D viewer

Hand of helix of a internal gear in 3D viewer is correctly shown.

### SP 1 - Housing deformation calculation when the shaft strength calculation is ignored

The housing deformation calculation does not run, if the shaft strength calculation is ignored in the control panel.

### SP 1 - Load spectrum requested lifetime with factors

When the requested lifetime is defined with factors in the load spectrum tab, the values are not changing anymore while changing the calculation type.

### SP 1 - Not loaded gears are ignored during load spectrum

Not loaded gears throughout a load spectrum calculation are now completely ignored.

### SP 1 - Own input losses can now be defined as torque or power

Own input losses can now be defined as torque or power, and not only as efficiency.

### SP 1 - Part lists do not show data when calculations were not run

Bearing and shaft list reports do not show data when shaft calculations are ignored, and gear list report does not show data when "Only kinematics" is selected.

### SP 1 - Sense of rotation for fixed shaft showing the proper direction

With a fixed shaft, the sense of rotation is now showing the correct direction, depending on the relative speed to the carrier.

### SP 1 - Update of the data of idle gears

The gear data of the idle gear in a multimesh gear calculation is now passed correctly to the shaft calculation, for the case of taking the gear body into account.

## Sketcher

### SP 1 - Annotation improvements

Several minor issues with annotations have been improved. The data update between annotations and element editor has also been improved.

### SP 1 - Mirroring face gears

Mirroring face gears is now possible.

## Forced Response Gears

### SP 1 - **Forced response analysis of the gear mesh outputs in the time domain**

Forced response analysis of the gear mesh outputs in the time domain is shown based on the last running speed.

## Gears

### SP 1 - **Bold print variant solution in gear sizing results**

Bold print indicator fixed for gear sizing result tables.

### SP 1 - **Considering tooth contact stiffness from tooth form in load spectrum**

Tooth contact stiffness from tooth form is now considered correctly in the load spectrum calculation.

### SP 1 - **Gear body overlaps with gear geometry in the 3D viewer.**

Fix of the overlap of the gear body and the gear geometry in the 3D viewer (in gear calculation, shaft calculation and KISSdesign).

### SP 1 - **Input of temperature in the load spectrum**

The input field for the temperature in the load spectrum table is now shown correctly.

### SP 1 - **Material properties of Duracon M90-44 from Polyplastics**

Root fatigue data is added for plastic material Duracon M90-44 from Polyplastics. Some general material properties are also updated.

### SP 1 - **Rounding for number of teeth**

Double value for number of teeth is rounded instead of truncated for the integer setting for the number of teeth.

### SP 1 - **The oil viscosity graph is shown correctly for small viscosity values**

The oil viscosity graph was now correctly drawn even when the oil viscosity is smaller than 10.

## Cylindrical Gears Rating

### SP 1 - **Improvement in the calculation of $Y_{CHD}$ and $Z_{CHD}$ factors calculated according to FVA 271**

$Y_{CHD}$  and  $Z_{CHD}$  factors calculated according to FVA 271 now properly consider the min/max hardening depth defined by the user.

### SP 1 - **Influence of alternating bending on the flank safety factor**

Tooth flank load spectrum setting "Calculate both cases and document the unfavorable case" works correctly in cases that  $Y_M$  factor for all gears is set to 1.0.

### SP 1 - **Scuffing results in the special Load spectrum report**

Scuffing results per bin are now shown again in the special load spectrum report.

## Cylindrical Gears Contact Analysis

### SP 1 - Comments added back to contact analysis graphics

The texts in the contact analysis graphics were missing and now are available in the comments window.

### SP 1 - Legends added back to contact analysis graphics

The legends were missing from the contact analysis 2D graphics and are now added back.

### SP 1 - Number of pitches is set based on pitch error

If the "Own input" is not selected for resolution, the number of pitches used in the contact analysis will always be 1, if there is no pitch error, and 2, if there is pitch error. Previously it didn't revert back to 1 when pitch error was removed.

## Cylindrical Gears Geometry

### SP 1 - Pinion type cutter dimension in tab Tooth form

Pinion type cutter dimension in tab Tooth form were not drawn correctly in case a full rounding or chamfer was set.

### SP 1 - Profile diagram shows individual modification curves properly

Profile diagram shows individual modification curves properly when the axis settings is set to roll angle, diameter and rotation angle.

### SP 1 - Step export of a planetary system that contains a planet carrier

Improved stability of the software during the step export of a planetary system that contains a planet carrier.

### SP 1 - Tool drawing with dimensions

Several improvements were made to the tool dimension drawing.

## Globoid Worm Gears

### SP 1 - The user input for $J_{0T}$ coefficient

The user input  $J_{0T}$  coefficient for wear calculation is now saved in the calculation file.

## Shafts

### SP 1 - Calculation of factor $K_{WK,RS}$ according to FKM

Formula for the calculation of factor  $K_{WK,RS}$  to estimate the fatigue limit of surface treated components according to FKM has been corrected.

### SP 1 - Internal message for beams fixed in shaft editor

Internal RotCAD message for beams has been fixed in the shaft editor.

### SP 1 - $K_{AK}$ factor was calculated with maximum equivalent stress instead of the equivalent stress per load bin

The  $K_{AK}$  factor per load bin was calculated with the maximum equivalent stress  $\sigma_{mv}$  and  $\tau_{mv}$  instead of the equivalent stress per load bin.

### SP 1 - Shaft solver convergence

Convergence of shaft solver for classic bearing calculation has been improved for some cases.

### SP 1 - Update of a gear calculation file connected to a shaft calculation file

Improvement for the case a shaft calculation file reads the gear data from a gear calculation file and something changes in this gear calculation file.

## Bearings

### SP 1 - Fatigue load limit for custom rolling bearing read from the file

Fatigue load limit  $C_u$  for custom rolling bearing, whose data was read from the calculation file, was not considered in the calculation of modified reference rolling bearing lifetime. Instead, internally calculated fatigue load limit was used.

### SP 1 - Rolling bearing damage for single load level and lifetime from load spectrum

When calculation with single load level and lifetime from load spectrum is chosen, rolling bearing damage calculation now considers lifetime from chosen load bin (before nominal required service lifetime was considered).

### SP 1 - Rolling bearing lifetime calculation with single bin calculation

Rolling bearing lifetime calculation for single bin with the lubricant temperature defined individually per bin has been improved.

### SP 1 - Rolling bearing table structure

The structure of the rolling bearing table has been improved.

## Bolts

### SP 1 - Typing error in formula of the guideline VDI 2230 paper 1 (2015) to calculate the length of engagement

Typing error in formula of the guideline VDI 2230 paper 1 (2015) to calculate the length of engagement (Bolt thread critical) is fixed. In formula 213 denominator  $D_1$  instead of  $d_{min}$  should be written.



## FEM Calculations

### SP 1 - Fine tune the FEM mesh density of big planet carrier models

Improve between runtime and accuracy of the FEM calculation for big planet carrier models.

### SP 1 - Improvement of the LTCA - FEM interface for gear body calculations

The internal interface of LTCA to FEM in case of gear body simulations was improved and made more robust. Appropriate user message was added.

### SP 1 - View of the gear body in the 3D graphic of the gear

The gear body was not shown in the 3D view of the gear after a gear body calculation.

## COM-Interface

### SP 1 - Running rough sizing through COM or skript

Running rough sizing for gears is possible again through COM or skript. The corresponding functions are `setupRoughSizing()` and `calculateRoughSizing()`.

## Scripting

### SP 1 - Graphic generation through SKRIPT

Improvement of image generation from SKRIPT. Robustness issues have been resolved.

### SP 1 - Stopping a script causes deletion of code

Bugfix of the editor losing the previous run code, when the running script was stopped manually through the editor.

## CAD-Interfaces

### SP 1 - Interface to Autodesk Inventor 2025

Interface to Autodesk Inventor 2025 added.

## Interfaces for Data Exchange

### SP 1 - Clean additional mass component attributes in the REXS\_Export function

Clean attributes in additional mass component in the REXS\_Export function of KISSsys.

### SP 1 - GDE export of design\_helix and design\_profile

Values in the GDE file for design\_helix and design\_profile are now exported in millimeter units.

### SP 1 - GDE export of internal gears

Measurement tolerances for  $M_{dk}$  are now exported with the correct sign for internal gears.

## KISSsys

### SP 1 - Database access error for shaft calculations with needle cage bearings

When trying to calculate KISSsys files created before release 2024, which contain needle cage rolling bearings with wrongly set clearance, a message with Database access error was being shown.

### SP 1 - Journal bearings' reaction forces

Reaction forces from the journal bearings are now transferred from the shaft calculation to the system calculation.

### SP 1 - Shaft axis alignment dialog

The selection of own input to calculate the axis alignment is now available.

### SP 1 - Spiral bevel gear setting

The setting for spiral bevel gear in the bevel gear calculation is transferred from KISSsoft to KISSsys.

### SP 1 - Support's reaction forces

Reaction forces from the supports transferred from the shaft calculation to the system calculation.